## THE LAGRANGIAN DERIVATION OF KANE'S EQUATIONS

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#### **ABSTRACT**

The Lagrangian approach to the development of dynamics equations for a multi-body system, constrained or otherwise, requires solving the forward kinematics of the system at velocity level in order to derive the kinetic energy of the system. The kinetic-energy expression should then be differentiated multiple times to derive the equations of motion of the system. Among these differentiations, the partial derivative of kinetic energy with respect to the system generalized coordinates is specially cumbersome. In this paper, we will derive this partial derivative using a novel kinematic relation for the partial derivative of angular velocity with respect to the system generalized coordinates. It will be shown that, as a result of the use of this relation, the equations of motion of the system are directly derived in the form of Kane's equations.

Keywords: dynamics modelling, Kane's equations, multi-body system, and virtual work.

## LA DÉRIVATION LAGRANGIENNE DES ÉQUATIONS DE KANE

## RÉSUMÉ

L'approche lagrangienne pour le développement des équations de dynamique pour un système multi-corps, contraint ou pas, exige la résolution de la cinématique directe du système au niveau des vitesses afin de déterminer l'énergie cinétique du système. L'expression de l'énergie cinétique doit alors être dérivée plusieurs fois afin de déterminer les équations du mouvement du système. Parmi ces dérivations, la dérivée partielle de l'énergie cinétique par apport aux coordonnées généralisées du système est particulièrement complexe. Dans cet article, nous dériverons cette dérivée partielle en utilisant une relation cinématique novatrice pour la dérivée partielle de la vitesse angulaire par rapport aux coordonnées généralisées du système. Nous montrerons que, en raison de l'utilisation de cette relation, les équations du mouvement du système sont directement dérivées sous forme d'équations de Kane.

Mots clés: modélisation dynamique, équations de Kane, système multi-corps, et travail virtuel.

#### 1 Introduction

Motivated by the need for the dynamics analysis of complex systems, many researchers have tried to develop the equations of motion of multi-body systems in novel forms more suitable for numerical computations, symbolic manipulations, or both. The Newton-Euler formalism, despite its strength, seemed unattractive because it requires the computation of all constraint forces and moments whereas energy-based methods, e.g., the Euler-Lagrange, disregard the constraint forces and moments based on the assumption that these forces and moments do not contribute to the total work performed. The Euler-Lagrange method, however, has its own issue that makes its application cumbersome. The issue is that it involves differentiating the kinetic and potential energies of the entire multi-body system with respect to the system generalized coordinates and velocities. These differentiations combined with the nonlinear nature of the relation between the twist of each body in the system to the system generalized coordinates render the "raw" application of the method to multi-body systems impractical. For instance, for a three-link planar mechanism with only three degrees of freedom, one has to write pages of equations to be able to properly compute the kinetic energy and differentiate it as needed [1].

In this paper, to simplify the differentiations of the system kinetic energy, we derive closed-form expressions for the partial derivatives of the translational and angular velocities of a body with respect to the system generalized coordinates and velocities. Among these relations, it is the partial derivative of the angular velocity with respect to the generalized coordinates that is new. This relation was derived a few years ago [2] for the purpose of obtaining the linearized kinematics of structurally flexible serial manipulators. The relation had the condition that the generalized coordinates of the system be *independent*. Moreover, the application of that relation to the derivation of dynamics equations was not discussed. In this paper, we use the above-mentioned expressions to differentiate the kinetic energy of an unconstrained system. Furthermore, we will extend that relation to the case of constrained systems and use the result to derive the equations of motion of constrained multi-body systems.

It will be demonstrated that, using the above-mentioned closed-form relations, the Euler-Lagrange formalism produces the equations of motion in the form of Kane's equations. Among all attempts to develop more efficient formulations for the equations of motion of multi-body systems, Kane's [3; 4] has proven to be probably, by far, the most controversial. The controversy, however, is not on the technical merits or the accuracy of Kane's final results; rather, it mostly concerns the originality of the equations, and the way they are obtained. Kane's equations have been compared with the earlier results of Gibbs and Appell [5], Jourdain [6] and Maggi [7]. For a brief summary of such discussions, see [8].

A fundamental issue is that Kane considers the concept of virtual displacement "objectionable" [9]. As such, avoiding D'Alembert's Principle, he starts from Newton's Second Law. Kane's equations have also been derived from "the work-energy form of Newton's Second Law" [10]. However, to derive the equations from the Second Law, whether in its original or in its work-energy form, one must eliminate the system constraint forces as they do not appear in Kane's equations. This is exactly where D'Alembert's Principle comes into the picture, as it is—in general—the virtual work of constraint forces which vanishes. (In fact, D'Alembert's Principle acts as a "physical postulate" independent from the Newton-Euler equations [11].) Here, basing our derivation on D'Alembert's principle of virtual work, we derive Kane's equations as a direct and natural consequence of the application of the Lagrange equations to multi-body systems.

In what follows, Kane's equations are very briefly introduced in Section 2. In Section 3, we will discuss the principle of virtual work and show how this principle can be used to derive the Lagrange equations for constrained and unconstrained multi-body systems. Section 4 is dedicated to the derivation of equations of motion for systems with tree structures. Section 5 discusses the same for constrained systems, the paper concluding with some remarks in Section 6.

#### 2 KANE'S EQUATIONS

For a system of n rigid bodies with r independent quasi-velocities  $u_i$ , Kane's equations are written as [4]

$$F_j + F_j^* = 0$$
 for  $j = 1, \dots, r$  (1)

In the above set of equations,  $F_j$  and  $F_j^*$  are the impressed and the inertial generalized forces, respectively. They are given by

$$F_{j} \triangleq \sum_{i=1}^{n} \left[ \left( \frac{\partial v_{i}}{\partial u_{j}} \right)^{T} f_{i} + \left( \frac{\partial \omega_{i}}{\partial u_{j}} \right)^{T} n_{i} \right]$$
 (2)

$$F_j^* \triangleq -\sum_{i=1}^n \left[ \left( \frac{\partial v_i}{\partial u_j} \right)^T m_i \dot{v}_i + \left( \frac{\partial \omega_i}{\partial u_j} \right)^T (I_i \dot{\omega}_i + \omega_i \times I_i \omega_i) \right]$$
(3)

In the above,  $v_i$  and  $\omega_i$  are the mass-centre translational velocity and the angular velocity of the *i*th body;  $f_i$  and  $n_i$  represent the resultants of the impressed forces and moments acting on the body; and  $m_i$  and  $I_i$  denote the body mass and moment of inertia.

#### 3 THE PRINCIPLE OF VIRTUAL WORK

The D'Alembert principle of virtual work in Lagrange's form [12; 11], which has also been called the Lagrange Principle [11], can be reformulated to obtain

$$\phi^T \delta q = 0$$
 where  $\phi \triangleq \frac{d}{dt} \left( \frac{\partial T}{\partial \dot{q}} \right) - \frac{\partial T}{\partial q} - f$  (4)

where q and  $\dot{q}$  are the vectors of generalized coordinates and velocities, respectively;  $\delta q$ , as usual, represents the virtual change in the generalized coordinates; and T denotes the system kinetic energy. The vector f of total *impressed* generalized forces in the above equation represents the active forces applied on the multi-body system. This force can be computed from the definition of virtual work:

$$\delta W = \sum_{i=1}^{n} \left( \mathbf{f}_{i}^{T} \frac{\partial \mathbf{v}_{i}}{\partial \dot{\mathbf{q}}} + \mathbf{n}_{i}^{T} \frac{\partial \omega_{i}}{\partial \dot{\mathbf{q}}} \right) \delta \mathbf{q} \equiv \mathbf{f}^{T} \delta \mathbf{q}$$
 (5)

where  $f_i$  and  $n_i$  are the resultant impressed force and moment applied on the *i*th body, respectively;  $v_i$  and  $\omega_i$  are the mass-centre translational velocity and the angular velocity of the *i*th body of the system.

When q is an n-dimensional vector of independent generalized coordinates of the system, all entries of  $\delta q$ , i.e., the virtual changes in the coordinates, will be independent and arbitrary. The Lagrange equation is then derived in vector form as

$$\phi \equiv \frac{d}{dt} \left( \frac{\partial T}{\partial \dot{q}} \right) - \frac{\partial T}{\partial q} - f = 0 \tag{6}$$

and the vector of generalized forces can be obtained as

$$f = \sum_{i=1}^{n} \left[ \left( \frac{\partial v_i}{\partial \dot{q}} \right)^T f_i + \left( \frac{\partial \omega_i}{\partial \dot{q}} \right)^T n_i \right]$$
 (7)

However, if the generalized coordinates are not independent, the virtual changes cannot be imposed independent of each other, and the equations of motion will not be as simple.

In the absolute majority of applications, the constraints can be written in the Pfaffian differential form as

$$A(q,t) dq = b(q,t) dt$$
 (8)

where A is assumed to be an  $r \times n$ -dimensional, full-rank matrix. Equivalently, the above constraint equation can be written as

$$A(q,t)\dot{q} = b(q,t) \tag{9}$$

Any vector that, when replacing  $\dot{q}$ , satisfies the above equation is termed an *admissible velocity*.

Virtual displacements have two properties: They are imposed instantly, i.e., in frozen time, and they must comply with the system constraints. The former property is by definition, and the latter is due to the principle [8]. Consequently, the virtual changes of the generalized coordinates should satisfy eq. (8) in the form below:

$$A(q,t)\,\delta q = 0\tag{10}$$

Geometrically, this equation means that the virtual change must lie in the null-space of the matrix A, i.e.,  $\delta q$  must be orthogonal to all rows of A. Hence, the dimension of the subspace within which  $\delta q$  can arbitrarily vary is  $m \triangleq n - r$ . On the other hand, according to the Lagrange Principle (4),  $\delta q$  must also be orthogonal to the n-dimensional vector  $\phi$ . This simply means that  $\phi$  has to lie in the subspace spanned by the rows of A, namely, its *row space*:

$$\phi \in \mathcal{R}(\mathbf{A}^T). \tag{11}$$

Because the system generalized coordinates and velocities are constrained by r independent constraint equations, only m of the generalized velocities will be independent. As such, we should be able to find an m-dimensional vector u—with independent entries—which can produce all admissible velocities  $\dot{q}$  through the  $n \times m$ -dimensional, full-rank matrix B(q,t) and the n-dimensional vector d(q,t) through

$$\dot{q} = B(q,t) u + d(q,t)$$
 with  $B(q,t) \triangleq \frac{\partial \dot{q}}{\partial u}$  (12)

The entries of u may or may not be time-derivatives themselves; those entries that belong to the latter category are called quasi-velocities, while the others are simply generalized velocities. In Kane's method terminology, the rows of B are called partial velocities, and the entries of u are known as generalized speeds [4].

Since  $\dot{q}$  has to be admissible, it must satisfy the constraint equation (9). Therefore, we should have

$$A(q,t)B(q,t)u + A(q,t)d(q,t) - b(q,t) = 0$$

for any given u. Consequently, we obtain

$$A(q,t) B(q,t) = \mathbf{O}_{r \times m} \quad \text{and} \quad A(q,t) d(q,t) = b(q,t)$$
(13)

The first equation shows that the range space of B is orthogonal to all vectors in the row space of A. Equivalently, since  $B^TA^T$  also vanishes, one can say that the row space of A is a subset of the null-space of  $B^T$ , namely,

$$\mathcal{R}(\mathbf{A}^T) \subset \mathcal{N}(\mathbf{B}^T) \tag{14}$$

From eqs. (11) and (14), we can conclude that

$$\phi \in \mathcal{N}(B^T)$$
 or  $B^T \phi = 0$ 

Hence, if we can find a complete set of independent generalized velocities, quasi-velocities, or both, such as the entries of u, then the Lagrange equations for the system can be written as

$$\left(\frac{\partial \dot{q}}{\partial u}\right)^{T} \phi \equiv \left(\frac{\partial \dot{q}}{\partial u}\right)^{T} \left[\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{q}}\right) - \frac{\partial T}{\partial q} - f\right] = 0 \tag{15}$$

The above m equations are independent from each other and fully express the dynamics of the system.

Similarly, we can derive the relation below for the generalized forces:

$$\left(\frac{\partial \dot{q}}{\partial u}\right)^{T} f = \left(\frac{\partial \dot{q}}{\partial u}\right)^{T} \sum_{i=1}^{n} \left[ \left(\frac{\partial v_{i}}{\partial \dot{q}}\right)^{T} f_{i} + \left(\frac{\partial \omega_{i}}{\partial \dot{q}}\right)^{T} n_{i} \right]$$
(16)

It should be noted that, when the generalized coordinates are dependent, the generalized force f cannot be derived uniquely from the definition of the virtual work, eq. (5). However, that does not pose a problem because, in fact, those are the components of f along the columns of f which are needed for eq. (15), and those can one compute uniquely from eq. (16).

<sup>&</sup>lt;sup>1</sup>The terms quasi-coordinates and quasi-velocities, however, go back to the beginning of the 20th century [13].

## 4 DYNAMICS OF MULTI-BODIES WITH A TREE STRUCTURE

In this case, one can readily choose the joint values as a set of independent generalized coordinates. Therefore, the Lagrange equation (6) can be written as

$$\frac{d}{dt}\left(\frac{\partial T}{\partial \dot{q}}\right) - \frac{\partial T}{\partial q} + \frac{\partial V}{\partial q} = f_{\text{nc}}$$
(17)

where the entries of q are the independent generalized coordinates of the system. In the above equation, the generalized force has been divided into conservative and nonconservative parts.  $f_{\rm nc}$  denotes the nonconservative part while its conservative counterpart is represented by the scalar function V of the system potential energy.

## 4.1 The kinetic energy

The kinetic energy of the system of n rigid-bodies can be computed as

$$T = \frac{1}{2} \sum_{i=1}^{n} \omega_i^T I_i \, \omega_i + \frac{1}{2} \sum_{i=1}^{n} v_i^T m_i v_i$$
 (18)

where  $\omega_i$  and  $I_i$  respectively are the angular velocity and the centroidal moment of inertia of the body, both expressed in the *i*th body frame;  $v_i$  and  $m_i$  are the velocity of the body mass-centre and the mass of the body, respectively. The mass-centre velocity is expressed in the inertial frame.

The angular and the translational velocities are, in general, nonlinear functions of the generalized coordinates of the system, q, and linear functions of the generalized velocities,  $\dot{q}$ . These functional relations can be established through the forward kinematics of the system. They will be of the form

$$\omega_i \equiv \omega_i(q, \dot{q}) = J_{\omega i} \dot{q}, \qquad \Rightarrow \qquad \dot{\omega}_i = \frac{d}{dt} (J_{\omega i} \dot{q}) = J_{\omega i} \ddot{q} + \dot{J}_{\omega i} \dot{q}$$
 (19)

$$v_i \equiv v_i(q, \dot{q}) = J_{vi} \dot{q} \qquad \Rightarrow \qquad \dot{v}_i = \frac{d}{dt} (J_{vi} \dot{q}) = J_{vi} \ddot{q} + \dot{J}_{vi} \dot{q}$$
 (20)

where, for simiplicity, we have assumed that the system is sceleronomic, and  $J_{wi}$  and  $J_{vi}$  are defined as

$$J_{\omega i} \triangleq \frac{\partial \omega_i}{\partial \dot{q}}$$
 and  $J_{vi} \triangleq \frac{\partial v_i}{\partial \dot{q}}$  (21)

Traditionally, at this stage, the kinetic-energy expression is expanded in terms of q and  $\dot{q}$  by using eqs. (19) and (20), and the expression is reformulated as<sup>2</sup>

$$T = \frac{1}{2} \dot{q}^T M \dot{q}$$

Thereafter, one has to work with the generalized inertia matrix M of the system, and differentiate the entries of that matrix with respect to q. Here, however, we will try to continue with the kinetic energy as expressed in eq. (18). To that end, we recall the following theorem [2]. A new proof of the theorem is produced in Appendix A.

**Theorem 1.** The partial derivative of the angular velocity  $\omega$  of a rigid body in a kinematic chain with respect to the chain independent generalized-coordinate vector  $\mathbf{q}$  can be expressed in terms of the Jacobian  $\mathbf{J}_{\omega}$ , its time rate, and the cross-product matrix  $[\omega \times]$  of the angular velocity as

$$\frac{\partial \omega}{\partial \mathbf{g}} = \dot{\mathbf{J}}_{\omega} + [\omega \times] \, \mathbf{J}_{\omega} \tag{22}$$

<sup>&</sup>lt;sup>2</sup>For example, one can refer to [14; 15; 1], among others.

Another relation that we will need in our derivation is the expression for the partial derivative of the velocity of the body mass-centre with respect to the system generalized coordinates:

$$\frac{\partial v_i}{\partial q} = \frac{\partial \dot{c}_i}{\partial q} = \frac{d}{dt} \left( \frac{\partial c_i}{\partial q} \right) = \frac{d}{dt} \left( \frac{\partial \dot{c}_i}{\partial \dot{q}} \right) = \frac{d}{dt} \left( \frac{\partial v_i}{\partial \dot{q}} \right)$$

$$\Rightarrow \frac{\partial v_i}{\partial q} = \dot{J}_{vi}$$
(23)

In the above equations,  $c_i$  denotes the position vector of the centre of mass of the *i*th body expressed in the fixed frame

Below, we will use eqs. (22) and (23) to derive all the terms of the Lagrange equation (6).

# Computing $\frac{d}{dt}(\partial T/\partial \dot{q})$

Using the relations derived above, We can compute the partial derivative of T with respect to the generalized velocities  $\dot{q}$  as

$$\frac{\partial T}{\partial \dot{q}} = \sum_{i=1}^{n} \left( \frac{\partial \omega_{i}}{\partial \dot{q}} \right)^{T} I_{i} \, \omega_{i} + \sum_{i=1}^{n} \left( \frac{\partial v_{i}}{\partial \dot{q}} \right)^{T} m_{i} v_{i}$$

$$= \sum_{i=1}^{n} J_{\omega_{i}}^{T} I_{i} J_{\omega_{i}} \, \dot{q} + \sum_{i=1}^{n} J_{vi}^{T} m_{i} J_{vi} \, \dot{q} \tag{24}$$

The right-hand side of eq. (24) can then be written in a more compact form:

$$\frac{\partial T}{\partial \dot{q}} = \sum_{i=1}^{n} J_i^T M_i J_i \, \dot{q} \tag{25}$$

where the Jacobian matrix  $J_i$  and the inertia dyad  $M_i$  of the *i*th link are defined as

$$J_i \triangleq \begin{bmatrix} J_{\omega i} \\ J_{vi} \end{bmatrix}$$
 and  $M_i \triangleq \begin{bmatrix} I_i & O \\ O & m_i 1 \end{bmatrix}$  (26)

It should be stressed that, as defined above, the two blocks  $J_{\omega i}$  and  $J_{vi}$  of  $J_i$  are referred to two different coordinate frames, namely, the body frame and the base frame, respectively. Moreover, because  $I_i$  is expressed in a body-attached frame, the inertia dyad remains constant throughout the motion of the multi-body system.

Finally, differentiating eq. (25) with respect to time, we have

$$\frac{d}{dt}(\frac{\partial T}{\partial \dot{q}}) = \sum_{i=1}^{n} J_{i}^{T} M_{i} \frac{d}{dt} (J_{i} \dot{q}) + \sum_{i=1}^{n} \dot{J}_{i}^{T} M_{i} J_{i} \dot{q}$$
(27)

#### Computing $\partial T/\partial q$

Equations (22) and (23) can be used to differentiate the kinetic-energy expression (18) with respect to the generalized coordinates:

$$\frac{\partial T}{\partial q} = \sum_{i=1}^{n} \left( \frac{\partial \omega_{i}}{\partial q} \right)^{T} I_{i} \omega_{i} + \sum_{i=1}^{n} \left( \frac{\partial v_{i}}{\partial q} \right)^{T} m_{i} v_{i}$$

$$= \sum_{i=1}^{n} \left( \dot{J}_{\omega i} + \left[ \omega_{i} \times \right] J_{\omega i} \right)^{T} I_{i} J_{\omega i} \dot{q} + \sum_{i=1}^{n} \dot{J}_{v i}^{T} m_{i} J_{v i} \dot{q} \tag{28}$$

which can be rewritten in the more compact form below:

$$\frac{\partial T}{\partial q} = \sum_{i=1}^{n} \left( \dot{J}_{i}^{T} M_{i} J_{i} - J_{i}^{T} W_{i} M_{i} J_{i} \right) \dot{q}$$
(29)

in which the skew-symmetric,  $6 \times 6$  angular-velocity dyad  $W_i$  is defined as

$$W_i \triangleq \begin{bmatrix} [\boldsymbol{\omega}_i \times] & \boldsymbol{O}_{3 \times 3} \\ \boldsymbol{O}_{3 \times 3} & \boldsymbol{O}_{3 \times 3} \end{bmatrix}$$
 (30)

## 4.2 The potential energy

The effect of conservative forces, e.g., gravity, can be properly accounted for using a potential energy function. For brevity, we only consider the gravitational potential energy here; if there are flexibilities in the system, the potential energy should be complemented with the elastic potential energy. The potential energy can therefore be expressed as

$$V = -\sum_{i=1}^{n} m_i c_i^T g \tag{31}$$

where g is the gravitational acceleration. Therefore, the partial derivative of the potential energy with respect to the generalized coordinates can be computed as

$$\frac{\partial V}{\partial \boldsymbol{q}} = -\sum_{i=1}^{n} m_i \left(\frac{\partial \boldsymbol{c}_i}{\partial \boldsymbol{q}}\right)^T \boldsymbol{g} = -\sum_{i=1}^{n} m_i \left(\frac{\partial \boldsymbol{v}_i}{\partial \dot{\boldsymbol{q}}}\right)^T \boldsymbol{g}$$

Then, the vector of conservative generalized forces can be obtained from

$$f_{c} \triangleq -\frac{\partial V}{\partial q} = \sum_{i=1}^{n} m_{i} J_{vi}^{T} g$$
(32)

Of course, one can also include gravity in the model by propagating it from the base of the chain upward, which amounts to the base having an acceleration of -g. For one such algorithm, one can refer to [16].

For large multi-bodies deployed on orbit, the vector of gravitational acceleration may vary from element to element due to the dependence of the gravitational acceleration on the distance of the link centre of mass from the Earth centre. If one or more of the bodies is so big that its centres of mass and gravity do not coincide, an extra term due to the effect of the resulting gravitational moment should also be added to the expression of the potential energy. In such a case, the weight of the body applies a moment known as the *gravity-gradient torque* [17] about the centre of mass of the body. These gravitational effects cannot be handled by the propagation method mentioned above because, in this case, the gravitational acceleration is a vector function of the system generalized coordinates, not just a constant vector.

### 4.3 Applied forces and moments

Let us assume that, in addition to the joint-actuation forces and moments, represented here by  $\tau$ , there are impressed external forces  $f_i^{\rm ex}$  and moments  $n_i^{\rm ex}$  applied on the bodies; these external forces are assumed to be applied at the mass centres. We further assume that both  $f_i^{\rm ex}$  and  $n_i^{\rm ex}$  are expressed in the body frame of the *i*th body. Then, the generalized force  $f_{\rm nc}$  applied on the system can be computed from

$$f_{\text{nc}} = \tau + \sum_{i=1}^{n} J_{i}^{T} w_{i}^{\text{ex}} \quad \text{where} \quad w_{i}^{\text{ex}} \triangleq \begin{bmatrix} \mathbf{1}_{3 \times 3} & \mathbf{O}_{3 \times 3} \\ \mathbf{O}_{3 \times 3} & R_{i} \end{bmatrix} \begin{bmatrix} \mathbf{B}_{i} n_{i}^{\text{ex}} \\ \mathbf{B}_{i} f_{i}^{\text{ex}} \end{bmatrix} \equiv \begin{bmatrix} \mathbf{B}_{i} n_{i}^{\text{ex}} \\ \mathbf{R} f_{i}^{\text{ex}} \end{bmatrix}$$
(33)

Rotation matrix  $R_i$  represents the rotation from the base-frame  $\Re$  to the body-frame  $\Im_i$ .

If there is damping in the joints, a generalized damping force  $f_d$  will be added to the right-hand side of the expression of  $f_{nc}$  in eq. (33).

## 4.4 The Lagrange equations

Substituting eqs. (27) and (28) in the left-hand side of Lagrange's equation (17), we obtain

$$\sum_{i=1}^{n} J_{i}^{T} M_{i} \frac{d}{dt} (J_{i} \dot{q}) + \sum_{i=1}^{n} \dot{J}_{i}^{T} M_{i} J_{i} \dot{q} - \sum_{i=1}^{n} (\dot{J}_{i}^{T} M_{i} J_{i} - J_{i}^{T} W_{i} M_{i} J_{i}) \dot{q} = f_{\text{nc}} + f_{c}$$

$$\Rightarrow \sum_{i=1}^{n} J_{i}^{T} \left[ M_{i} \frac{d}{dt} (J_{i} \dot{q}) + W_{i} M_{i} J_{i} \dot{q} \right] = f$$
(34)

where f is the sum of conservative and nonconservative impressed generalized forces. From eq. (7) and the definition of  $J_i$  in eq. (26), we can readily see that

$$f = \sum_{i=1}^{n} J_{i}^{T} w_{i} \quad \text{where} \quad w_{i} \triangleq \begin{bmatrix} \mathbf{1}_{3 \times 3} & \mathbf{O}_{3 \times 3} \\ \mathbf{O}_{3 \times 3} & R_{i} \end{bmatrix} \begin{bmatrix} \mathbf{B}_{i} n_{i} \\ \mathbf{B}_{i} f_{i} \end{bmatrix} \equiv \begin{bmatrix} \mathbf{B}_{i} n_{i} \\ \mathbf{R} f_{i} \end{bmatrix}$$
(35)

Therefore, the equations of motion can be written as

$$\sum_{i=1}^{n} J_i^T \left[ M_i \frac{d}{dt} (J_i \dot{q}) + W_i M_i J_i \dot{q} \right] = \sum_{i=1}^{n} J_i^T w_i$$
(36)

The part of the above equation within the brackets includes the inertial parts of the Newton and the Euler equations because

$$M_{i}\frac{d}{dt}(J_{i}\dot{q}) + W_{i}M_{i}J_{i}\dot{q} = \begin{bmatrix} I_{i}\dot{\omega}_{i} + \omega_{i} \times I_{i}\omega_{i} \\ m_{i}\dot{v}_{i} \end{bmatrix}$$
(37)

Since u can be chosen to be  $\dot{q}$  when the entries of q are independent, eq. (36), in effect, is the same as the set of Kane's equations—given by eqs. (1-3)—for the multi-body system with tree structure.

On the other hand, eq. (34) can be rewritten as

$$\sum_{i=1}^n \boldsymbol{J}_i^T \boldsymbol{M}_i \boldsymbol{J}_i \ddot{\boldsymbol{q}} + \sum_{i=1}^n \boldsymbol{J}_i^T \big( \boldsymbol{M}_i \dot{\boldsymbol{J}}_i + \boldsymbol{W}_i \boldsymbol{M}_i \boldsymbol{J}_i \big) \, \dot{\boldsymbol{q}} = \boldsymbol{f}$$

which can be simplified to

$$M(q)\ddot{q} + h(q,\dot{q}) = f_{\rm nc} + f_{\rm c} \tag{38}$$

$$M \triangleq \sum_{i=1}^{n} J_i^T M_i J_i \quad \text{and} \quad h(q, \dot{q}) \triangleq \left[ \sum_{i=1}^{n} J_i^T (M_i \dot{J}_i + W_i M_i J_i) \right] \dot{q}$$
 (39)

The positive-definite, symmetric matrix M is the system generalized inertia matrix. If taken to the other side of the equation, vector  $h(q, \dot{q})$  will represent the vector of centrifugal and Coriolis generalized forces.

Let us define a matrix C as a function of q and  $\dot{q}$  as

$$C(q, \dot{q}) \triangleq \sum_{i=1}^{n} J_i^T (M_i \dot{J}_i + W_i M_i J_i + M_i W_i J_i)$$
(40)

It can readily be seen that, because  $J_i\dot{q}$  lies in the null-space of  $W_i$ 

$$C(q,\dot{q})\dot{q} \equiv \sum_{i=1}^{n} J_{i}^{T} \left( M_{i}\dot{J}_{i} + W_{i}M_{i}J_{i} + M_{i}W_{i}J_{i} \right) \dot{q} \equiv \sum_{i=1}^{n} J_{i}^{T} \left( M_{i}\dot{J}_{i} + W_{i}M_{i}J_{i} \right) \dot{q} \equiv h(q,\dot{q})$$
(41)

Hence, eq. (38) can be reformulated in the form below:

$$M\ddot{q} + C\dot{q} = f_{\rm nc} + f_c \tag{42}$$

Matrix C, as defined in eq. (40), has the interesting property that  $\dot{M}-2C$  is skew-symmetric. For verification, we notice that

$$\begin{split} \dot{M} - 2C &= \sum_{i=1}^{n} \left( \dot{\boldsymbol{J}}_{i}^{T} \boldsymbol{M}_{i} \boldsymbol{J}_{i} + \boldsymbol{J}_{i}^{T} \boldsymbol{M}_{i} \dot{\boldsymbol{J}}_{i} \right) - 2 \sum_{i=1}^{n} \boldsymbol{J}_{i}^{T} \left( \boldsymbol{M}_{i} \dot{\boldsymbol{J}}_{i} + \boldsymbol{W}_{i} \boldsymbol{M}_{i} \boldsymbol{J}_{i} + \boldsymbol{M}_{i} \boldsymbol{W}_{i} \boldsymbol{J}_{i} \right) \\ &= \left[ \sum_{i=1}^{n} \left( \dot{\boldsymbol{J}}_{i}^{T} \boldsymbol{M}_{i} \boldsymbol{J}_{i} - \boldsymbol{J}_{i}^{T} \boldsymbol{M}_{i} \dot{\boldsymbol{J}}_{i} \right) \right] - 2 \left[ \sum_{i=1}^{n} \boldsymbol{J}_{i}^{T} \left( \boldsymbol{W}_{i} \boldsymbol{M}_{i} \boldsymbol{J}_{i} + \boldsymbol{M}_{i} \boldsymbol{W}_{i} \boldsymbol{J}_{i} \right) \right] \end{split}$$

which is the sum of two skew-symmetric matrices, thus being skew-symmetric itself. It should be noted that the number of floating-point operations required for solving eq. (42) is more than that for eq. (38); however, the former equation has application in proving the stability of different robot control schemes; see [14; 18; 19] for some examples.

#### 5 DYNAMICS OF CONSTRAINED MULTI-BODIES

Comparing eqs. (6) and (15) of Section 3, one can see that there is one essential difference between the dynamics equations of unconstrained and constrained systems. In constrained systems, it is the orthogonal projection of  $\phi$  along  $\partial \dot{q}/\partial u_i$  directions for  $i=1,\dots,m$  which vanish, not  $\phi$  itself:

$$\left(\frac{\partial \dot{q}}{\partial u}\right)^{T} \left[\frac{d}{dt} \left(\frac{\partial T}{\partial \dot{q}}\right) - \frac{\partial T}{\partial q} + \frac{\partial V}{\partial q}\right] = \left(\frac{\partial \dot{q}}{\partial u}\right)^{T} f_{nc}$$
(43)

Reviewing the developments reported in Section 4, one realizes that we have used the assumption of independent generalized coordinates in two locations: the differentiation of kinetic energy with respect to the generalized coordinates—where we used Theorem 1—and the derivation of the generalized forces, both conservative and nonconservative. From these two, the latter has already been addressed through equation (16). The former, on the other hand, will be discussed below.

For our purposes, the computation of  $\partial T/\partial q$  hinged on Theorem 1, which provides the partial derivative of the angular velocity of a body within a kinematic chain with respect to the chain independent generalized coordinates. Where the generalized coordinates are not independent, we can use the following result:

**Theorem 2.** The variation of the angular-velocity vector  $\omega$  of a rigid body in a kinematic chain due to a virtual change  $\delta q$  in the chain generalized coordinates can be expressed in terms of the Jacobian  $J_{\omega}$ , its time rate, and the cross-product matrix  $[\omega \times]$  of the angular velocity as

$$\left(\frac{\partial \omega}{\partial q}\right) \delta q = \left(\dot{J}_{\omega} + \left[\omega \times\right] J_{\omega}\right) \delta q \tag{44}$$

For the proof, one can refer to Appendix A.

Hence, if the generalized coordinates are subject to the Pfaffian constraints of eq. (8), we can invoke a reasoning similar to that used in Section 3 to show that—for an independent, complete set u of generalized velocities, quasi-velocities, or both—we will have

$$\left(\frac{\partial \dot{q}}{\partial u}\right)^{T} \left(\frac{\partial \omega}{\partial a}\right) = \left(\frac{\partial \dot{q}}{\partial u}\right)^{T} \left(\dot{J}_{\omega} + [\omega \times] J_{\omega}\right) \tag{45}$$

Substituting eqs. (45) and (27) in eq. (43), we will obtain

$$\left(\frac{\partial \dot{q}}{\partial u}\right)^{T} \sum_{i=1}^{n} J_{i}^{T} \left[ M_{i} \frac{d}{dt} (J_{i} \dot{q}) + W_{i} M_{i} J_{i} \dot{q} \right] = \left(\frac{\partial \dot{q}}{\partial u}\right)^{T} f$$
(46)

We can rewrite the left-hand side of the above equation as

LHS = 
$$\sum_{i=1}^{n} \left( J_{i} \frac{\partial \dot{q}}{\partial u} \right)^{T} \left[ M_{i} \frac{d}{dt} (J_{i} \dot{q}) + W_{i} M_{i} J_{i} \dot{q} \right]$$

$$= \sum_{i=1}^{n} T_{i}^{T} \left[ M_{i} \frac{d}{dt} (J_{i} \dot{q}) + W_{i} M_{i} J_{i} \dot{q} \right]$$
(47)

where matrix  $T_i$  is defined as

$$T_{i} \triangleq J_{i} \frac{\partial \dot{q}}{\partial u} \equiv \begin{bmatrix} \frac{\partial \omega_{i}}{\partial \dot{q}} \frac{\partial \dot{q}}{\partial u} \\ \frac{\partial v_{i}}{\partial \dot{q}} \frac{\partial \dot{q}}{\partial u} \end{bmatrix} \equiv \begin{bmatrix} \frac{\partial \omega_{i}}{\partial u} \\ \frac{\partial v_{i}}{\partial u} \end{bmatrix}$$
(48)

On the other hand, the right-hand side of eq. (46) can be rewritten as

RHS = 
$$\left(\frac{\partial \dot{q}}{\partial u}\right)^T \sum_{i=1}^n \left(\left(\frac{\partial v_i}{\partial \dot{q}}\right)^T f_i + \left(\frac{\partial \omega_i}{\partial \dot{q}}\right)^T n_i\right) = \sum_{i=1}^n \left(J_i \frac{\partial \dot{q}}{\partial u}\right)^T w_i = \sum_{i=1}^n T_i^T w_i$$
 (49)

Therefore, the dynamics equations can be written as

$$\sum_{i=1}^{n} T_i^T \left[ M_i \frac{d}{dt} (J_i \dot{q}) + W_i M_i J_i \dot{q} \right] = \sum_{i=1}^{n} T_i^T w_i$$
(50)

As seen from eq. (48),  $T_i$  is a matrix composed of the partial derivatives of the angular and translational velocities<sup>3</sup> of the *i*th body with respect to the set of independent velocities u. Hence, the equation is in the form of Kane's equations (1-3).

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#### 6 CONCLUSIONS

Dynamics modelling of both constrained and unconstrained multi-body systems using the Euler-Lagrange approach was discussed in this paper. The approach requires differentiating the kinetic energy of the system with respect to the system generalized coordinates and velocities and subsequently differentiating the latter with respect to time. In the literature, the partial derivative of kinetic energy with respect to the generalized coordinates has been related to the partial derivative of the elements of the mass matrix with respect to the same variables. Due to the complicated relation between the elements of the mass matrix and these coordinates, the closed-form formulation of the dynamics model of a multi-body system through Lagrangian approach traditionally stops at this point.

In this paper, we derived all the relevant partial derivatives in closed form. The partial derivative of particular interest was that of the kinetic energy with respect to the system generalized coordinates, particularly the part of the kinetic energy pertaining to the rotational motion. Written in body-attached frames, the link inertia tensors become constants, thus leaving body angular velocities as the only variables. As such, we derived the partial derivative of the angular velocity with respect to the system generalized coordinates, in both cases of unconstrained and constrained systems, in closed form. Subsequently, the dynamics equations of the system were derived; the result was the equations of motion of the system in the form of Kane's equations.

#### APPENDIX A

**Theorem 1.** The partial derivative of the angular velocity  $\omega$  of a rigid body in a kinematic chain with respect to the chain independent generalized-coordinate vector  $\mathbf{q}$  can be expressed in terms of the Jacobian  $\mathbf{J}_{\omega}$ , its time rate, and the cross-product matrix  $[\omega \times]$  of the angular velocity as

$$\frac{\partial \omega}{\partial q} = \dot{J}_{\omega} + [\omega \times] J_{\omega} \tag{A-1}$$

*Proof:* The angular velocity of a body can be expressed as a function of the body Euler parameters  $\eta$  and their time derivatives, i.e., as  $\omega \equiv \omega(\eta, \dot{\eta})$ . More specifically, as seen from eq. (B-19), in the body frame, we have

$$\tilde{\omega} = 2\dot{\boldsymbol{\eta}} \otimes \boldsymbol{\eta}^* \equiv 2\boldsymbol{\eta}^* \circledast \dot{\boldsymbol{\eta}} \tag{A-2}$$

<sup>&</sup>lt;sup>3</sup>Notice that the angular velocity and the translational velocity are expressed in two different frames, the former in a body-attached frame and the latter in an inertial frame. As such, the array containing  $\omega_i$  and  $v_i$  is not technically the *twist*.

where  $\tilde{\omega} \triangleq [\omega^T \ 0]^T$  is the augmented angular-velocity vector, and  $\eta^*$  is the conjugate of  $\eta$ , as defined in item 3 of Appendix B. Here, we have used the properties of the quaternion composition operators  $\circledast$  and  $\otimes$ . The definitions of these operators along with some of their properties are given in Appendix B.

Differentiating eq. (A-2) with respect to the generalized coordinates q, we can use the chain rule to obtain

$$\begin{split} \frac{\partial \tilde{\omega}}{\partial q} &= \frac{\partial \tilde{\omega}}{\partial \eta} \frac{\partial \eta}{\partial q} + \frac{\partial \tilde{\omega}}{\partial \dot{\eta}} \frac{\partial \dot{\eta}}{\partial q} = 2 \eta^* \circledast \frac{\partial \dot{\eta}}{\partial q} + 2 \dot{\eta} \otimes \frac{\partial \eta^*}{\partial q} \\ &= 2 \eta^* \circledast \frac{d}{dt} (\frac{\partial \dot{\eta}}{\partial \dot{q}}) - 2 \dot{\eta} \otimes \eta^* \otimes (\eta^* \circledast \frac{\partial \eta}{\partial q}) \end{split}$$

where we have used eq. (B-17), and that

$$\eta \equiv \eta(q) \quad \Rightarrow \quad \frac{\partial \eta}{\partial q} \equiv \frac{\partial \dot{\eta}}{\partial \dot{q}} \quad \text{and} \quad \frac{\partial \dot{\eta}}{\partial q} \equiv \frac{d}{dt} \left(\frac{\partial \dot{\eta}}{\partial \dot{q}}\right)$$

Therefore, using eq. (B-18), we can see that

$$\begin{split} \frac{\partial \tilde{\omega}}{\partial q} &= 2\boldsymbol{\eta}^* \circledast \frac{d}{dt} (\frac{1}{2}\boldsymbol{\eta} \circledast \frac{\partial \tilde{\omega}}{\partial \dot{q}}) - 2(\dot{\boldsymbol{\eta}} \otimes \boldsymbol{\eta}^*) \otimes \left[\boldsymbol{\eta}^* \circledast (\frac{1}{2}\boldsymbol{\eta} \circledast \frac{\partial \tilde{\omega}}{\partial \dot{q}})\right] \\ &= \boldsymbol{\eta}^* \circledast \left[\dot{\boldsymbol{\eta}} \circledast \frac{\partial \tilde{\omega}}{\partial \dot{q}} + \boldsymbol{\eta} \circledast \frac{d}{dt} (\frac{\partial \tilde{\omega}}{\partial \dot{q}})\right] - \frac{1}{2} \tilde{\omega} \otimes \frac{\partial \tilde{\omega}}{\partial \dot{q}} \\ &= \frac{1}{2} \tilde{\omega} \circledast \frac{\partial \tilde{\omega}}{\partial \dot{q}} + \frac{d}{dt} (\frac{\partial \tilde{\omega}}{\partial \dot{q}}) - \frac{1}{2} \tilde{\omega} \otimes \frac{\partial \tilde{\omega}}{\partial \dot{q}} \\ \Rightarrow & \frac{\partial \tilde{\omega}}{\partial \boldsymbol{q}} = \frac{d}{dt} (\frac{\partial \tilde{\omega}}{\partial \dot{q}}) + \frac{1}{2} (\left[\tilde{\omega} \circledast\right] - \left[\tilde{\omega} \otimes\right]) \frac{\partial \tilde{\omega}}{\partial \dot{q}} \end{split} \tag{A-3}$$

From the last equation, we can immidiately see that

$$\frac{\partial \omega}{\partial q} = \dot{J}_{\omega} + [\omega \times] J_{\omega} \tag{A-4}$$

in which  $J_{\omega}$  is defined by

$$J_{\omega} \triangleq \frac{\partial \omega}{\partial \dot{a}} \tag{A-5}$$

and  $[\omega \times]$  represents the cross-product matrix of  $\omega$ .

Similarly, one can show that a similar relation holds in the inertial frame.

**Corollary 1.** The partial derivative of the angular velocity  ${}^{\mathfrak{R}}\omega$ , where  ${\mathfrak{R}}$  represents the inertial frame, of a rigid body in a kinematic chain with respect to the chain independent generalized-coordinate vector  ${\bf q}$  can be expressed in terms of the Jacobian  ${}^{\mathfrak{R}}{\bf J}_{\omega}$ , its time-rate, and the cross-product matrix  $[{}^{\mathfrak{R}}\omega\times]$  of the angular velocity as

$$\frac{\partial^{\mathfrak{R}} \omega}{\partial q} = {}^{\mathfrak{R}} \dot{J}_{\omega} - [{}^{\mathfrak{R}} \omega \times] {}^{\mathfrak{R}} J_{\omega}. \tag{A-6}$$

**Theorem 2.** The variation of the angular-velocity vector  $\omega$  of a rigid body in a kinematic chain due to a virtual change in the chain generalized coordinates q can be expressed in terms of the Jacobian  $J_{\omega}$ , its time rate, and the cross-product matrix  $[\omega \times]$  of the angular velocity as

$$\left(\frac{\partial \omega}{\partial q}\right) \delta q = \left(J_{\omega} + \left[\omega \times\right] J_{\omega}\right) \delta q \tag{A-7}$$

*Proof:* This theorem can essentially be proven the same way as Theorem 1. The only difference is that, because the generalized coordinates can be dependent, eq. (B-17) no longer holds. However, we can still relate the virtual changes  $\delta \eta^*$  and  $\delta \eta$  through

$$\frac{\partial \eta^*}{\partial q} \delta q = -\eta^* \otimes \left( \eta^* \circledast \frac{\partial \eta}{\partial q} \right) \delta q \tag{A-8}$$

which is basically the same equation projected along  $\delta q$ .

Hence, we will have

$$\left(\frac{\partial \omega}{\partial q}\right)\delta q = \left(\dot{J}_{\omega} + \left[\omega \times\right] J_{\omega}\right)\delta q$$

#### APPENDIX B

Let us consider two rotations represented by quaternions  $\eta_1$  and  $\eta_2$  and, at the same time, by rotation matrices  $Q_1$  and  $Q_2$ , respectively.

1. If these two rotations are performed one after the other in such a way that the resultant rotation  $Q_3$  is given by  $Q_3 = Q_1Q_2$ , the compound quaternion  $\eta_3$  is given by

$$\eta_3 = \eta_1 \circledast \eta_2 \equiv \eta_2 \otimes \eta_1 \tag{B-9}$$

where quaternion multiplication operations  $\circledast$  and  $\otimes$  are defined as below [20]:

$$[\eta \circledast] \triangleq \begin{bmatrix} \eta_0 \mathbf{1} + [\eta_v \times] & \eta_v \\ -\eta_v^T & \eta_0 \end{bmatrix}, \quad \text{and} \quad [\eta \otimes] \triangleq \begin{bmatrix} \eta_0 \mathbf{1} - [\eta_v \times] & \eta_v \\ -\eta_v^T & \eta_0 \end{bmatrix}$$
 (B-10)

in which  $\eta_v$  and  $\eta_0$  are the vector and scalar parts of the quaternion, respectively.

2. The two quaternion composition operators have associative properties, i.e.,

$$(\eta_1 \circledast \eta_2) \circledast \eta_3 \equiv \eta_1 \circledast (\eta_2 \circledast \eta_3) \tag{B-11}$$

$$(\eta_1 \otimes \eta_2) \otimes \eta_3 \equiv \eta_1 \otimes (\eta_2 \otimes \eta_3) \tag{B-12}$$

These properties can readily be verified in a symbolic manipulation software such as Maple.

3. Any quaternion  $\boldsymbol{\eta} \triangleq \begin{bmatrix} \boldsymbol{\eta}_v^T & \eta_0 \end{bmatrix}^T$  has a unique conjugate  $\boldsymbol{\eta}^* \triangleq \begin{bmatrix} -\boldsymbol{\eta}_v^T & \eta_0 \end{bmatrix}^T$ , so that

$$\eta \circledast \eta^* \equiv \eta^* \circledast \eta \equiv \eta \otimes \eta^* \equiv \eta^* \otimes \eta \equiv \begin{bmatrix} 0 & 0 & 0 & 1 \end{bmatrix}^T$$
(B-13)

The partial derivatives of  $\eta$  and  $\eta^*$  with respect to the set of generalized coordinates are related through

$$\frac{\partial \eta^*}{\partial q} = -D \frac{\partial \eta}{\partial q} \tag{B-14}$$

where matrix D is defined as

$$D \triangleq \begin{bmatrix} \mathbf{1}_{3 \times 3} & \mathbf{0}_3 \\ \mathbf{0}_3^T & -1 \end{bmatrix} \tag{B-15}$$

It can readily be shown that D has the following properties

$$D \eta \circledast \equiv \eta^* \otimes D$$
 and  $D \eta \otimes \equiv \eta^* \circledast D$  (B-16)

If the generalized coordinates are independent, then we will also have

$$\frac{\partial \eta^*}{\partial q} = -\eta^* \otimes \left(\eta^* \circledast \frac{\partial \eta}{\partial q}\right) \tag{B-17}$$

4. The time-derivative of the quaternion in the body-attached frame is given by [20]

$$\dot{\boldsymbol{\eta}} = \frac{1}{2}\,\tilde{\boldsymbol{\omega}} \otimes \boldsymbol{\eta} \equiv \frac{1}{2}\,\boldsymbol{\eta} \circledast \tilde{\boldsymbol{\omega}} \tag{B-18}$$

where the augmented angular velocity of the body is defined as  $\tilde{\omega} \triangleq \begin{bmatrix} \omega^T & 0 \end{bmatrix}^T$ . Using the properties 1, 2, and 4 above, we can solve eq. (B-18) for  $\tilde{\omega}$  as

$$\tilde{\omega} = 2\dot{\eta} \otimes \eta^* \equiv 2\eta^* \circledast \dot{\eta} \tag{B-19}$$

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